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# A Tutorial on Fractional Order Motion Control

Part III: Fractional Order Position Servo

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- Fractional Order Motion Controls

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- Dr. Ying Luo

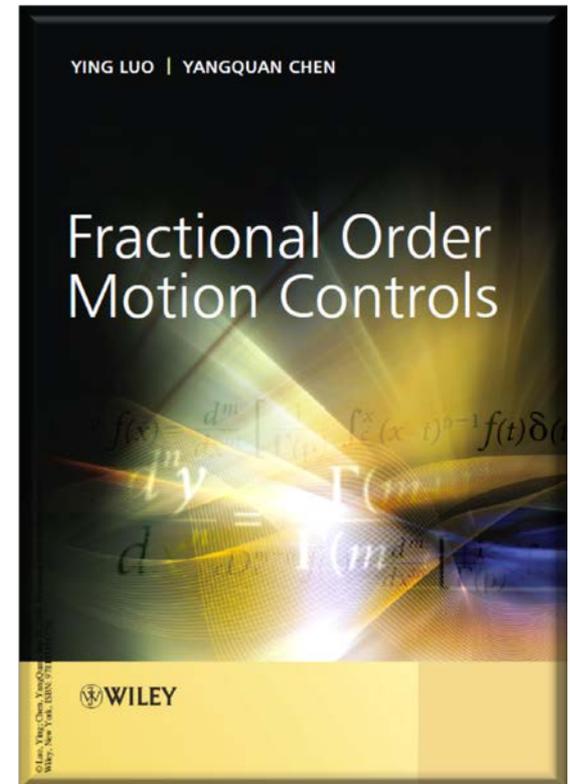
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## The model to be discussed

Second order position servo

$$P(s) = \frac{1}{s(Ts + 1)}$$

## Gain and phase in frequency domain

$$\angle P(j\omega) = -\tan^{-1}(\omega T) - \frac{\pi}{2}$$
$$|P(j\omega)| = \frac{1}{\omega\sqrt{1 + (\omega T)^2}}$$

## The traditional integer order PD controller

$$C(s) = K_p(1 + K_d s)$$

## Integer order PD controller design using “flat phase” concept

$$G(s) = P(s)C(s) = \frac{K_p(1 + K_d s)}{s(Ts + 1)}$$

$$\angle G(j\omega) = \arctan(\omega K_d) - \arctan(\omega T)$$

$$\left. \frac{d\angle G(j\omega)}{d\omega} \right|_{\omega=\omega_c} = \frac{K_d}{1 + (K_d \omega_c)^2} - \frac{T}{1 + (T \omega_c)^2}$$

$$\Rightarrow K_d = \frac{1}{T \omega_c^2}$$

This means: given a  $\omega_c$ , the phase margin at the flat part is fixed. No way to adjust !!

The fractional order proportional and derivative controller  $PD^\mu$

$$C(s) = K_p(1 + K_d s^\mu)$$

Gain and phase in frequency domain

$$\angle C(j\omega) = \tan^{-1} \frac{\sin \frac{(1-\mu)\pi}{2} + K_d \omega^\mu}{\cos \frac{(1-\mu)\pi}{2}} - \frac{(1-\mu)\pi}{2}$$

$$|C(j\omega)| = K_p \sqrt{\left(1 + K_d \omega^\mu \cos \frac{\mu\pi}{2}\right)^2 + \left(K_d \omega^\mu \sin \frac{\mu\pi}{2}\right)^2}$$

## Formulas for determining the parameters of FO PD controller

$$K_d = \frac{1}{\omega_c^\mu} \tan[\phi + \tan^{-1}(\omega_c T) - \frac{\mu\pi}{2} + \pi] \cos \frac{(1-\mu)\pi}{2} \quad (7.1)$$

$$-\frac{1}{\omega_c^\mu} \sin \frac{(1-\mu)\pi}{2}.$$

$$K_d = \frac{-B \pm \sqrt{B^2 - 4A^2\omega_c^{2\mu}}}{2A\omega_c^{2\mu}}, \quad (7.2)$$

$$A = \frac{T}{1 + (\omega_c T)^2},$$

$$B = 2A\omega_c^\mu \sin \frac{(1-\mu)\pi}{2} - \mu\omega_c^{\mu-1} \cos \frac{(1-\mu)\pi}{2}.$$

$$\begin{aligned} & |G(j\omega_c)| \\ &= |C(j\omega_c)||P(j\omega_c)| \\ &= \frac{K_p \sqrt{(1 + K_d\omega_c^\mu \cos \frac{\mu\pi}{2})^2 + (K_d\omega_c^\mu \sin \frac{\mu\pi}{2})^2}}{\omega_c \sqrt{1 + (\omega_c T)^2}} \\ &= 1. \end{aligned} \quad (7.2)$$

### The procedure to obtain the controller parameters

- (1) Given  $\omega_c$ , the gain crossover frequency.
- (2) Given  $\Phi_m$ , the desired phase margin.
- (3) Plot the curve 1,  $K_d$  with respect to  $\mu$ , according to (6.11).
- (4) Plot the curve 2,  $K_d$  with respect to  $\mu$ , according to (6.14).
- (5) Obtain the  $\mu$  and  $K_d$  from the intersection point on the above two curves.
- (6) Calculate the  $K_p$  from (6.15).

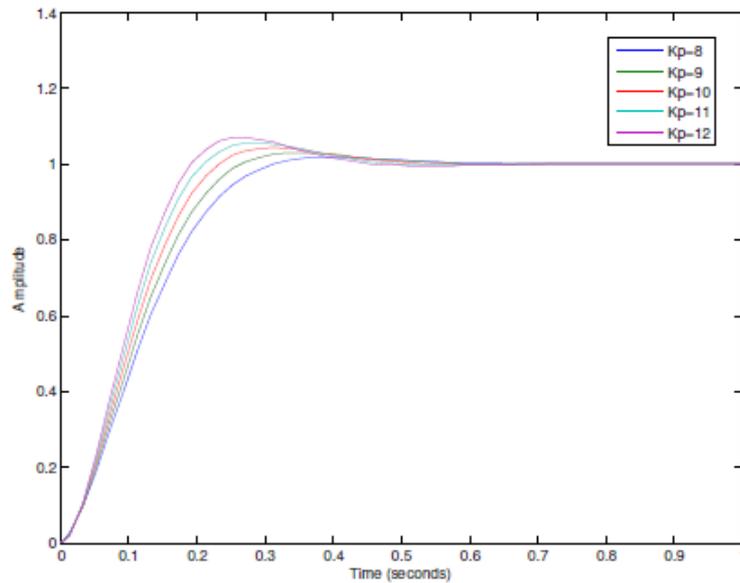


Figure: Step responses with the ITAE optimum proportional controller

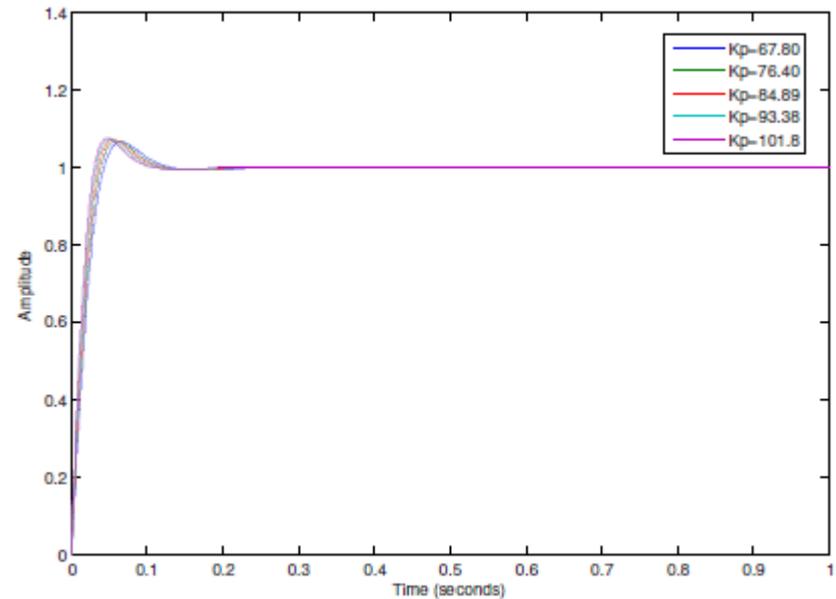


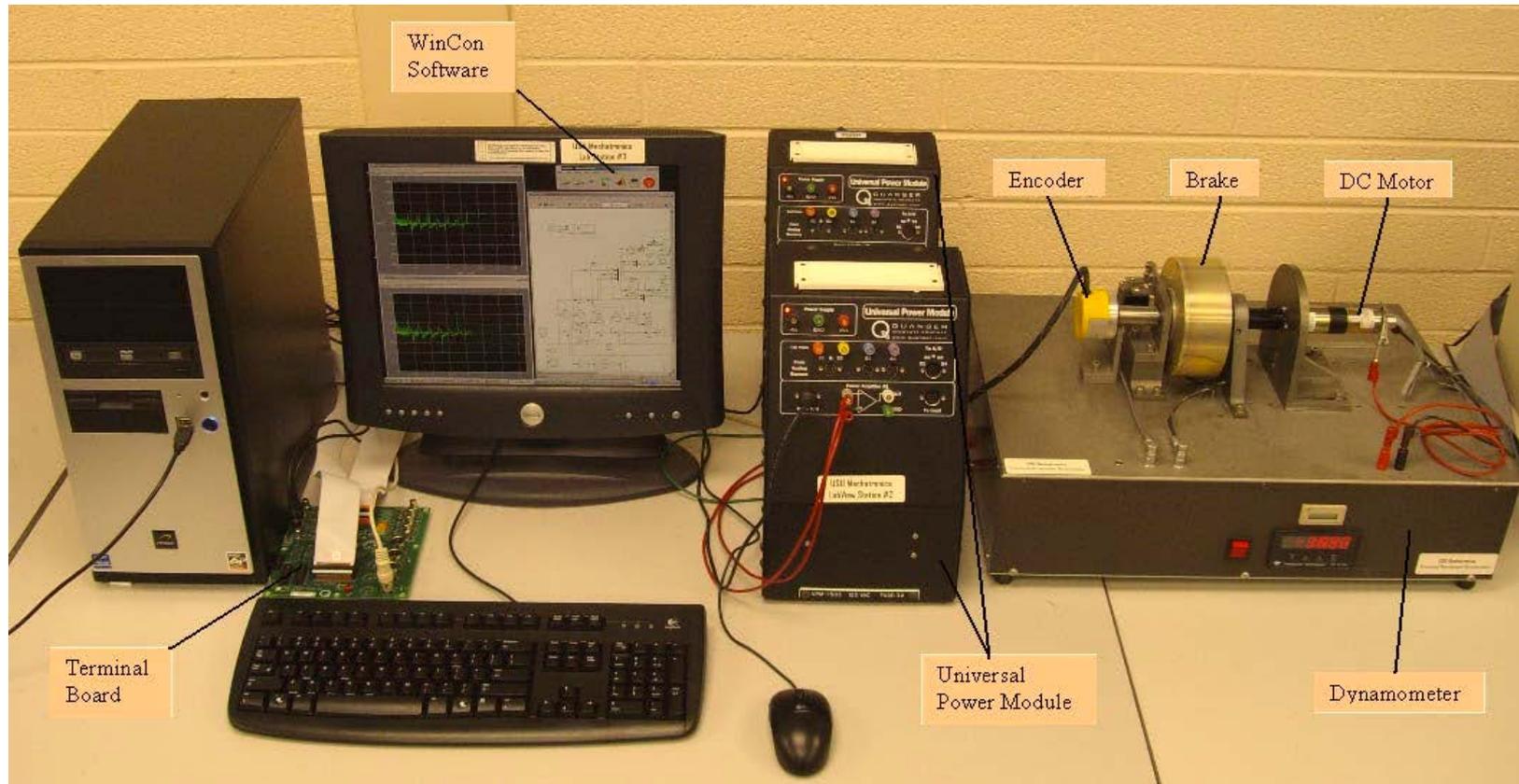
Figure: Step responses with  $PD^\lambda$  controller.

## The experimental platform

A general purpose fractional horsepower dynamometer

Quanser DAQ

Matlab/Simulink



## The model of the platform

Second order position servo

$$P(s) = \frac{1.52}{s(0.4s + 1)}$$

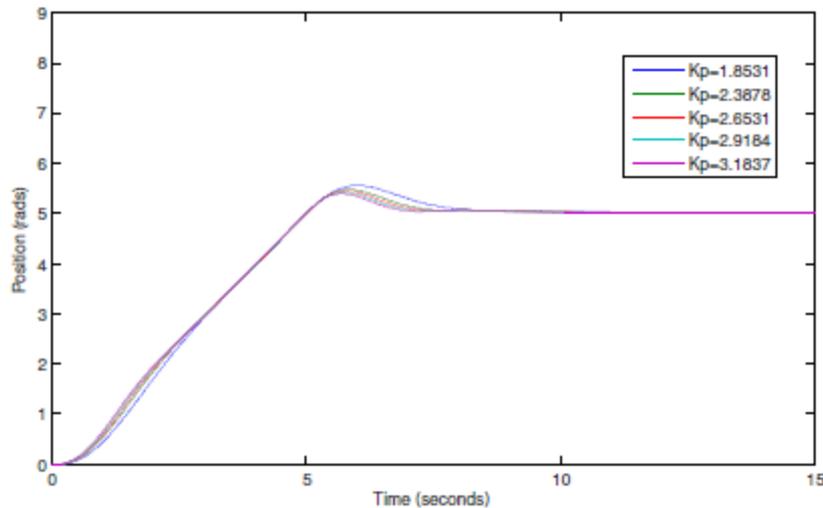


Figure: Step position responses with the ITAE optimal proportional controller

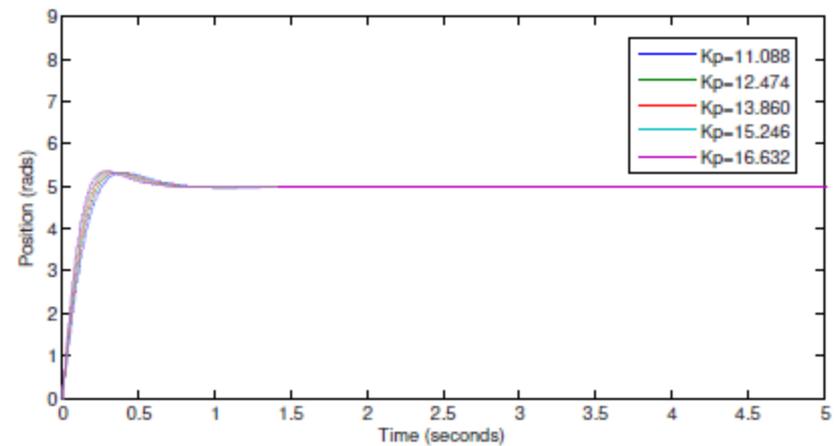
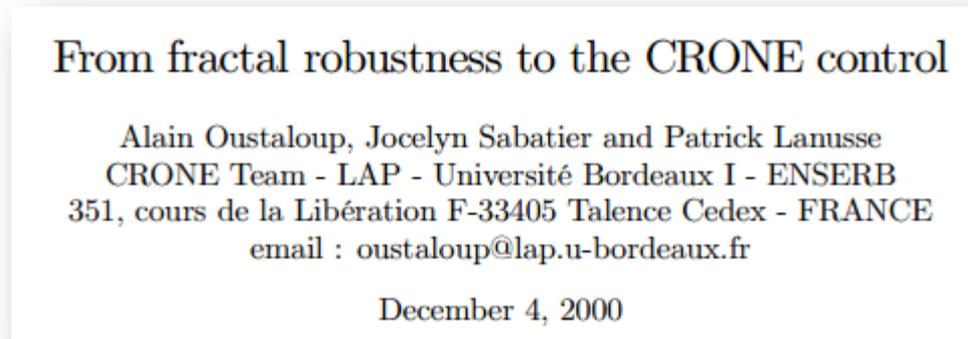
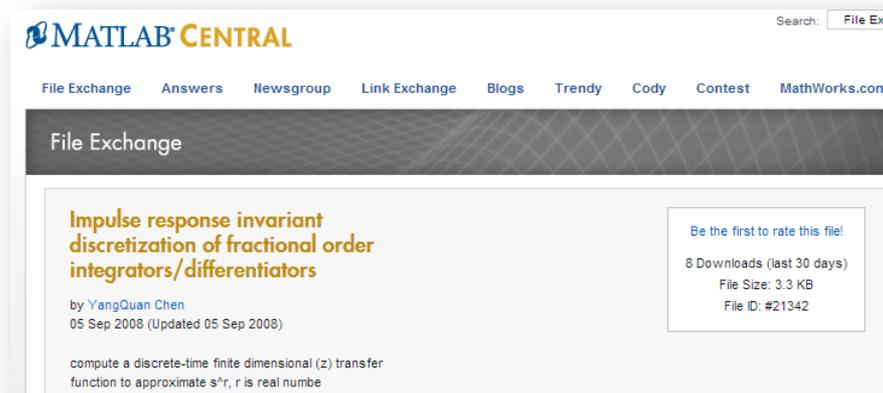


Figure: Step responses with  $PD^\lambda$  controller.

- Some of the methods to implement  $s^\lambda$
- The CRONE toolbox - Alain Oustaloup



- Impulse response invariant discretization (IRID) – Yangquan Chen



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## The model to be discussed

Second order position servo

$$P(s) = \frac{1}{s(Ts + 1)}$$

The fractional order [PD] controller  $(PD)^\mu$ 

$$C(s) = K_p(1 + K_d s)^\mu$$

## Gain and phase in frequency domain

$$\angle C(j\omega) = \mu \tan^{-1}(\omega K_d)$$

$$|C(j\omega)| = K_p [1 + (K_d \omega)^2]^{\frac{\mu}{2}}$$

## Some basic techniques for dealing with complex numbers

$$z = x + jy = |z|e^{j\varphi}$$

$$z^\alpha = (x + jy)^\alpha = (|z|e^{j\varphi})^\alpha = |z|^\alpha e^{j\varphi\alpha}$$

$$|z| = \sqrt{x^2 + y^2}$$

$$|z^\alpha| = |z|^\alpha = (\sqrt{x^2 + y^2})^\alpha$$

$$\angle z^\alpha = \varphi\alpha$$

## The “flat phase” tuning equations

$$\begin{aligned}
 |G_3(j\omega_c)| &= |C_3(j\omega_c)||P(j\omega_c)| \\
 &= K_{p3} \frac{(1 + (K_{d3}\omega_c)^2)^{\frac{\mu}{2}}}{\sqrt{(T\omega_c^2)^2 + \omega_c^2}} \\
 &= 1. \\
 \left. \begin{aligned}
 \frac{d(\angle(G_3(j\omega)))}{d\omega} \Big|_{\omega=\omega_c} &= \frac{\mu K_{d3}}{1 + (K_{d3}\omega_c)^2} - \frac{T}{1 + (T\omega_c)^2} \\
 &= 0, \\
 \angle[G_3(j\omega)] \Big|_{\omega=\omega_c} &= \mu \tan^{-1}(\omega_c K_{d3}) - \tan^{-1}(\omega_c T) - \frac{\pi}{2} \\
 &= -\pi + \phi_m.
 \end{aligned} \right\}
 \end{aligned}$$

## The procedure to obtain the controller parameters

- (1) Given  $\omega_c$ , the gain crossover frequency.
- (2) Given  $\Phi_m$ , the desired phase margin.
- (3) Plot the curve 1,  $K_{d3}$  with respect to  $\mu$ , according to (7.13).
- (4) Plot the curve 2,  $K_{d3}$  with respect to  $\mu$ , according to (7.16).
- (5) Obtain the  $K_{d3}$  and  $\mu$  from the intersection point on the above two curves.
- (6) Calculate the  $K_{p3}$  from (7.17).

## Implementation of FO [PD]

- The FO operator  $(1 + \tau s)^\mu$  for the FO [PD] controller can be implemented by modifying the code of the IRID.
- A discrete-time finite dimensional z transfer function is computed to approximate a continuous-time fractional order low-pass filter  $\frac{1}{(\tau s + 1)^\mu}$
- Implement  $\frac{1}{(\tau s + 1)^\mu}$ ,  $\mu \in (0, 1)$  first
- Then change  $\mu \in (-1, 0)$

- Plant model

$$P(s) = \frac{1}{s(0.4s + 1)}$$

- Design specifications

$$\omega_c = 10 \text{ rad/s}$$

$$\Phi_m = 70^\circ$$

Flat phase at  $\omega_c$

- IOPID parameters

$$K_p = 23.078, K_d = 0.102, K_i = -4.625$$

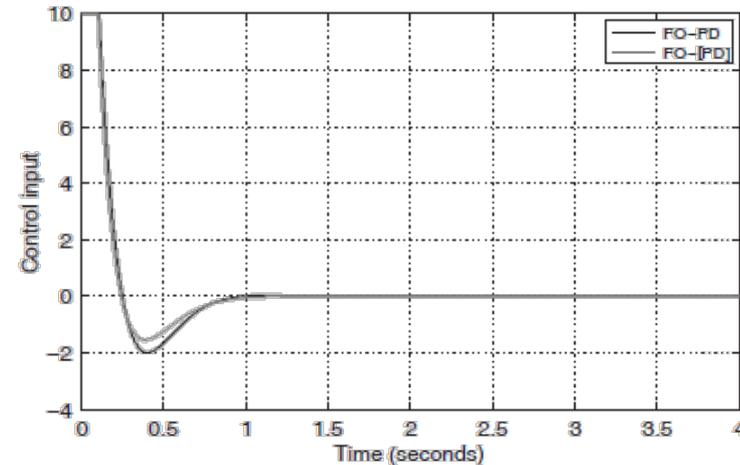
Unstable

- FO PD

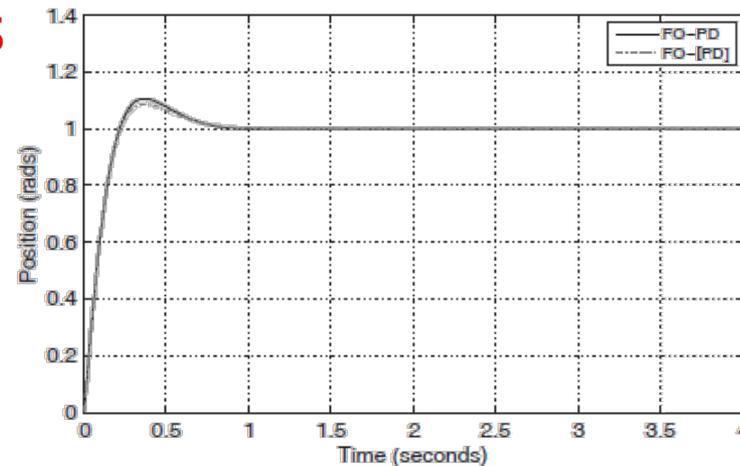
$$K_p = 16.784, K_d = 0.368, \lambda = 0.835$$

- FO [PD]

$$K_p = 13.860, K_d = 0.299, \lambda = 0.783$$



Simulation. Control input signals with two FO controllers ( $T = 0.4s$ )



Simulation. Step responses comparison with two FO controllers ( $T = 0.4s$ )

- Plant model

$$P(s) = \frac{1}{s(0.04s + 1)}$$

- Design specifications

$$\omega_c = 10 \text{ rad/s}$$

$$\Phi_m = 70^\circ$$

Flat phase at  $\omega_c$

- IOPID parameters

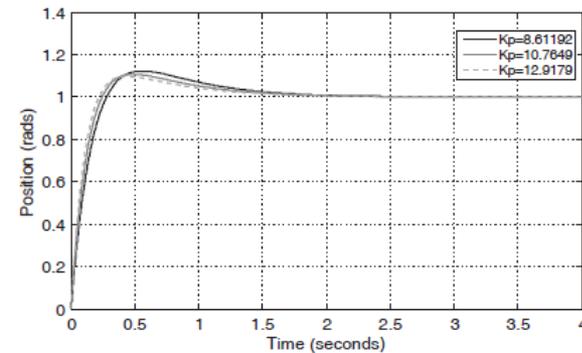
$$K_p = 10.76, K_d = 0.018, K_i = 1.567$$

- FO PD

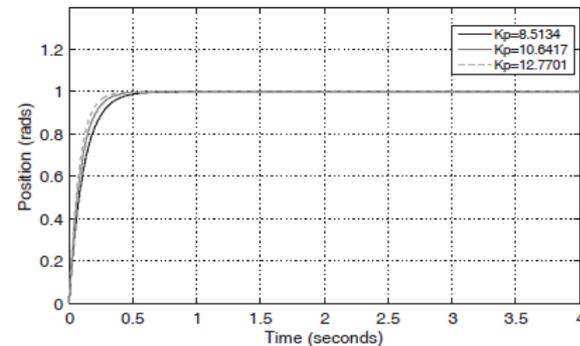
$$K_p = 10.64, K_d = 0.005, \lambda = 0.779$$

- FO [PD]

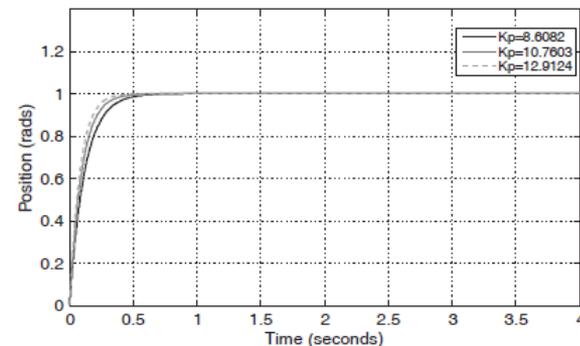
$$K_p = 10.76, K_d = 0.006, \lambda = 0.508$$



7.11 Simulation. Step responses with IOPID controller ( $T = 0.04s$ )



7.12 Simulation. Step responses with FOPD controller ( $T = 0.04s$ )



7.13 Simulation. Step responses with FO(PD) controller ( $T = 0.04s$ )

- Plant with time delay

$$P(s) = \frac{1}{s(0.4s + 1)} e^{-\tau s}$$

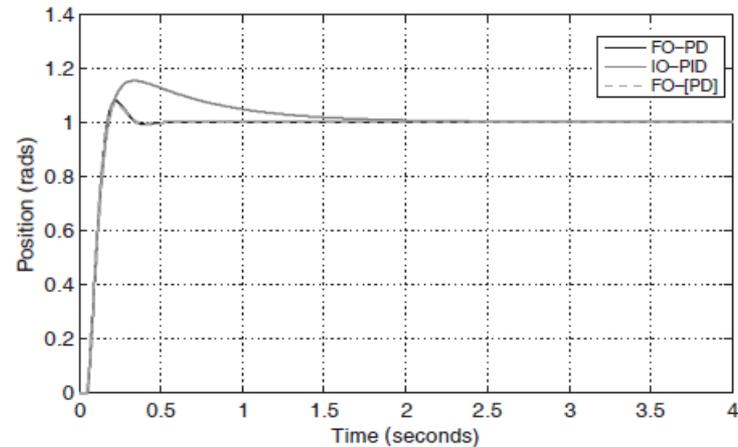
- Design specifications

$$\omega_c = 10 \text{ rad/s}$$

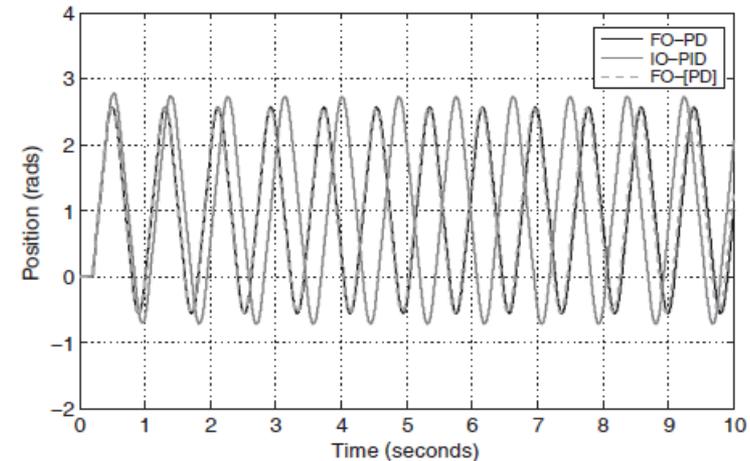
$$\Phi_m = 70^\circ$$

Flat phase at  $\omega_c$

- Plant with backlash nonlinearity



Simulation. Step responses comparison with time delay of 0.05s ( $T = 0.04s$ )



Simulation. Step responses comparison with time delay of 0.2s ( $T = 0.04s$ )

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### Basic idea of time-Constant robustness

Use the gradient of the phase margin to time constant  
and the gradient of the gain crossover frequency to time constant

## The plant to be controlled

Second order position servo with time delay

$$P(s) = \frac{1}{s(Ts + 1)} e^{-Ls}$$

The fractional order [PD] controller  $(PD)^\alpha$ 

$$C(s) = (K_p + K_d s)^\alpha$$

## The open loop transfer function

$$G(s) = C(s)P(s)$$

## Phase margin

$$\angle C(j\omega) = \alpha \arctan\left(\frac{K_d \omega_c}{K_p}\right) - \arctan(T \omega_c) - \frac{\pi}{2} - L \omega_c = \phi - \pi$$

## Gain crossover frequency specification

$$|G(j\omega)| = \frac{[(K_d \omega_c)^2 + K_p^2]^{\alpha/2}}{\omega_c \sqrt{1 + T^2 \omega_c^2}} = 1$$

## Robustness to time-constant variations

How ?

## Quantitatively evaluation of the robustness

$$-\frac{\frac{\partial |G(j\omega)|}{\partial \omega}}{\frac{\partial |G(j\omega)|}{\partial T}} \bigg|_{(\omega_{cs}, T_0)} = -\frac{\frac{\partial \angle[G(j\omega)]}{\partial \omega}}{\frac{\partial \angle[G(j\omega)]}{\partial T}} \bigg|_{(\omega_{cs}, T_0)} .$$

# Why ?

Gain robustness to  $\omega$  and  $T$ 

$$\left. \frac{\partial |G(j\omega)|}{\partial \omega} \right|_{(\omega_c, T_0)} \Delta\omega + \left. \frac{\partial |G(j\omega)|}{\partial T} \right|_{(\omega_c, T_0)} \Delta T = 0,$$

$$\frac{\Delta\omega}{\Delta T} = - \left. \frac{\frac{\partial |G(j\omega)|}{\partial \omega}}{\frac{\partial |G(j\omega)|}{\partial T}} \right|_{(\omega_c, T_0)},$$

Phase margin robustness to  $\omega$  and  $T$ 

$$\left. \frac{\partial \angle[G(j\omega)]}{\partial \omega} \right|_{(\omega_c, T_0)} \Delta\omega + \left. \frac{\partial \angle[G(j\omega)]}{\partial T} \right|_{(\omega_c, T_0)} \Delta T = 0,$$

$$\frac{\Delta\omega}{\Delta T} = - \left. \frac{\frac{\partial \angle[G(j\omega)]}{\partial \omega}}{\frac{\partial \angle[G(j\omega)]}{\partial T}} \right|_{(\omega_c, T_0)},$$

- The derivation for computation of the solution existence range

$$\begin{aligned} & \left. \frac{\partial |G(j\omega)|}{\partial \omega} \right|_{(\omega_c, T_0)} \\ &= \frac{(K_d^2 \omega_c^2 + K_p^2)^{\frac{\alpha}{2}} \alpha K_d^2}{(K_d^2 \omega_c^2 + K_p^2) (1 + T_0^2 \omega_c^2)^{\frac{1}{2}}} - \frac{(K_d^2 \omega_c^2 + K_p^2)^{\frac{\alpha}{2}}}{\omega_c^2 (1 + T_0^2 \omega_c^2)^{\frac{1}{2}}} - \frac{(K_d^2 \omega_c^2 + K_p^2)^{\frac{\alpha}{2}} T_0^2}{(1 + T_0^2 \omega_c^2)^{\frac{3}{2}}}, \end{aligned}$$

$$\left. \frac{\partial |G(j\omega)|}{\partial T} \right|_{(\omega_c, T_0)} = - \frac{(K_d^2 \omega_c^2 + K_p^2)^{\frac{\alpha}{2}} T_0 \omega_c}{(1 + T_0^2 \omega_c^2)^{\frac{3}{2}}}.$$

$$\left. \frac{\partial A}{\partial \omega} \right|_{(\omega_c, T_0)} = \frac{\alpha K_d}{K_p \left( 1 + \frac{K_d^2 \omega_c^2}{K_p^2} \right)} - \frac{T_0}{1 + T_0^2 \omega_c^2} - L,$$

$$\left. \frac{\partial A}{\partial T} \right|_{(\omega_c, T_0)} = - \frac{\omega_c}{1 + T_0^2 \omega_c^2}.$$

## Simplified two equations

$$\alpha \arctan(A) - \arctan(T_0 \omega_c) + \pi/2 - L\omega_c - \phi = 0, \quad (8.20)$$

$$(T_0 L \omega_c^2 - 1 + \alpha) A^2 - \alpha T_0 \omega_c A + T_0 L \omega_c^2 - 1 = 0. \quad (8.21)$$

## Solution for 8.20

$$A_{01} = \frac{\alpha \omega_c T_0 + \sqrt{\Delta_1}}{2(L \omega_c T_0 \omega_c - 1 + \alpha)},$$

$$A_{02} = \frac{\alpha \omega_c T_0 - \sqrt{\Delta_1}}{2(L \omega_c T_0 \omega_c - 1 + \alpha)},$$

$$\Delta_1 = (\alpha \omega_c T_0)^2 - 4(L \omega_c T_0 \omega_c - 1 + \alpha)(L \omega_c T_0 \omega_c - 1).$$

## Solution for 8.21

$$A_1 = \tan\left(\frac{\phi'}{\alpha}\right),$$

## Three cases

$$LT_0\omega_c^2 > 1$$

$$LT_0\omega_c^2 = 1$$

$$LT_0\omega_c^2 < 1$$

## 3 plants with different time constants

$$P_1(s) = \frac{1}{s(0.0241s + 1)} e^{-0.01s},$$

$$P_2(s) = \frac{1}{s(0.0265s + 1)} e^{-0.01s},$$

$$P_3(s) = \frac{1}{s(0.0301s + 1)} e^{-0.01s}.$$

## The resulting controller and implementation

$$C(s) = (160.5248 + 4.2539s)^{0.8332}.$$

$$C(z) = \frac{160.5z^5 - 486z^4 + 547.6z^3 - 278z^2 + 59.83z - 3.732}{0.05676z^5 - 0.1163z^4 + 0.07297z^3 - 0.01014z^2 - 0.00302z + 0.0004798}.$$

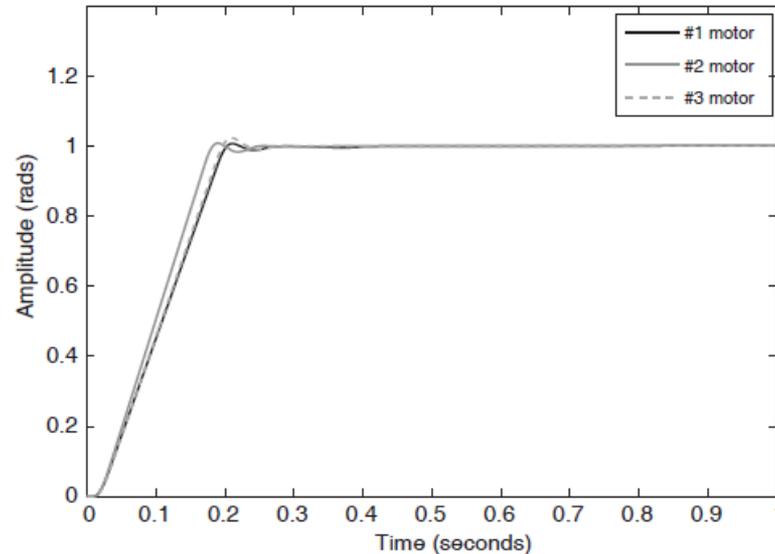


Figure: Step response comparison of the three motors models with different time constant

## Conclusion

The overshoots on the three motors are all less than 2%.  
Therefore, this tuning method guarantees system dynamic performance.

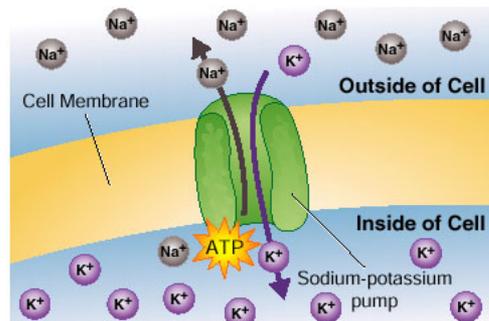
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## The plant to be controlled

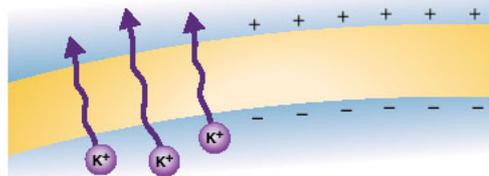
Fractional order model of *membrane charging*

Also: Fractional order position system

$$P(s) = \frac{1}{s(Ts^\alpha + 1)} e^{-Ls}$$



**A** A protein pump in the neuron cell membrane uses the energy of ATP to pump Na<sup>+</sup> out of the cell, and at the same time to pump K<sup>+</sup> in.



**B** The cell membrane is leakier to K<sup>+</sup> than it is to Na<sup>+</sup>. Because more positive charges leak out of the cell than leak in, the inside of the cell becomes negatively charged with respect to the outside.

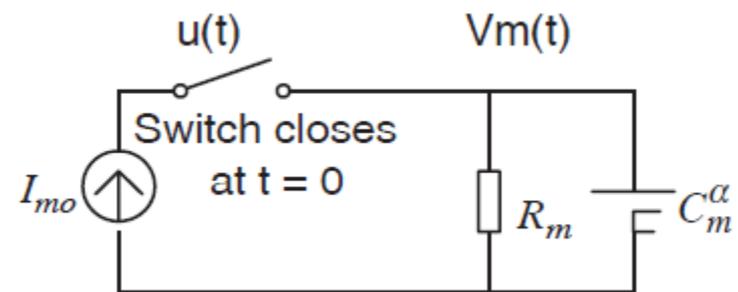


Figure: Membrane-charging circuit

$$C_m^\alpha \frac{d^\alpha V_m(t)}{dt^\alpha} + \frac{V_m(t)}{R_m} = I_{m0} u(t).$$

Figure: Cell membrane charging

Source:

<http://www.millerandlevine.com/chapter/35/898-899-rewrite.html>

## Time domain solution using Mittag-Leffler function

$$V_m(t) = I_{mo} R_m \frac{t^\alpha}{T} E_{\alpha, \alpha+1} \left( -\frac{t^\alpha}{T} \right).$$



G. M. Mittag-Leffler

## Still using the FO PD controller

Refer to slide 7

$$C(s) = K_p(1 + K_d s^\mu)$$

## Still using the “flat phase” design specification

$$\begin{aligned} \angle[G(j\omega_c)] &= \tan^{-1} \frac{\sin \frac{(1-\mu)\pi}{2} + K_d \omega_c^\mu}{\cos \frac{(1-\mu)\pi}{2}} \\ &\quad + \frac{\mu\pi}{2} - \frac{\pi}{2} + \tan^{-1} \frac{1 + T\omega_c^\alpha \cos \frac{\alpha\pi}{2}}{T\omega_c^\alpha \sin \frac{\alpha\pi}{2}} \end{aligned}$$

$$= -\pi + \phi_m,$$

$$|G(j\omega_c)| = \frac{K_p \sqrt{(1 + K_d \omega_c^\mu \cos \frac{\mu\pi}{2})^2 + (K_d \omega_c^\mu \sin \frac{\mu\pi}{2})^2}}{\sqrt{(T\omega_c^{1+\alpha} \sin \frac{\alpha\pi}{2})^2 + (\omega_c + T\omega_c^{1+\alpha} \cos \frac{\alpha\pi}{2})^2}} = 1.$$

$$\left. \frac{d(\angle(G(j\omega)))}{d\omega} \right|_{\omega=\omega_c} = \frac{K_d}{1 + (K_d \omega_c)^2} - \frac{\alpha T \sin \frac{\alpha\pi}{2} \omega_c^{\alpha-1}}{1 + T^2 \omega_c^{2\alpha} + 2T\omega_c^\alpha \cos \frac{\alpha\pi}{2}} = 0,$$

## The design procedure

- (1) Given  $\omega_c$ , the gain crossover frequency.
- (2) Given  $\phi_m$ , the desired phase margin.
- (3) Plot the curve 1,  $K_d$  with respect to  $\mu$ , according to (9.11).
- (4) Plot the curve 2,  $K_d$  with respect to  $\mu$ , according to (9.14).
- (5) Obtain the  $\mu$  and  $K_d$  from the intersection point on the above two curves.
- (6) Calculate  $K_p$  from (9.12).

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## The plant to be controlled

FO position system

$$P(s) = \frac{1}{s(Ts^\alpha + 1)}$$

## Gain and phase of the plant

$$\angle[P(j\omega)] = -\arctan \frac{T\omega^\alpha \sin\left(\frac{\alpha\pi}{2}\right)}{1 + T\omega^\alpha \cos\left(\frac{\alpha\pi}{2}\right)} - \frac{\pi}{2}$$

$$|P(j\omega)| = \frac{1}{\sqrt{[T\omega^{1+\alpha} \sin\left(\frac{\alpha\pi}{2}\right)]^2 + [\omega + T\omega^{1+\alpha} \cos\left(\frac{\alpha\pi}{2}\right)]^2}}$$

Still using the FO [PD] controller  $(PD)^\alpha$

$$C(s) = (K_p + K_d s)^\alpha$$

Still using the “flat phase” design specification

$$\angle[G_3(j\omega_c)] = \mu \tan^{-1}(\omega_c K_{d3}) - \tan^{-1} \left( \frac{T \omega_c^\alpha \sin \frac{\alpha\pi}{2}}{1 + T \omega_c^\alpha \cos \frac{\alpha\pi}{2}} \right) - \frac{\pi}{2}$$

$$= -\pi + \phi_m,$$

$$|G_3(j\omega_c)| = |C_3(j\omega_c)| |P(j\omega_c)|$$

$$= \frac{K_{p3} (1 + (K_{d3} \omega_c)^2)^{\frac{\mu}{2}}}{N}$$

$$= 1,$$

$$\left. \frac{d(\angle(G_3(j\omega)))}{d\omega} \right|_{\omega=\omega_c} = \frac{\mu K_{d3}}{1 + (K_{d3} \omega_c)^2} - \frac{\alpha T \omega_c^{\alpha-1} \sin \frac{\alpha\pi}{2}}{(T \omega_c^\alpha \sin \frac{\alpha\pi}{2})^2 + (1 + T \omega_c^\alpha \cos \frac{\alpha\pi}{2})^2}$$

$$= 0,$$

## The design procedure

- (1) Given parameters of the fractional order system to be controlled  $\alpha$  and  $T$ .
- (2) Given  $\omega_{c,r}$ , the gain crossover frequency.
- (3) Given  $\phi_{m,r}$  the desired phase margin.
- (4) Plot the curve 1,  $K_{d\beta}$  with respect to  $\mu$ , according to (10.10).
- (5) Plot the curve 2,  $K_{d\beta}$  with respect to  $\mu$ , according to (10.14).
- (6) Obtain  $K_{d\beta}$  and  $\mu$  from the intersection point on the above two curves.
- (7) Calculate the  $K_{p\beta}$  from (10.11).

- Plant model

$$P(s) = \frac{1}{s(0.4s^{1.4} + 1)}$$

- Design specifications

$$\omega_c = 10 \text{ rad/s}$$

$$\Phi_m = 70^\circ$$

Flat phase at  $\omega_c$

- IOPID parameters

$$Kp = 18.29, Kd = -0.0846, Ki = 42.45$$

Unstable

- FO PD

$$Kp = 10.916, Kd = 0.61, \lambda = 1.189$$

- FO [PD]

$$Kp = 6.31, Kd = 0.9435, \lambda = 1.2$$

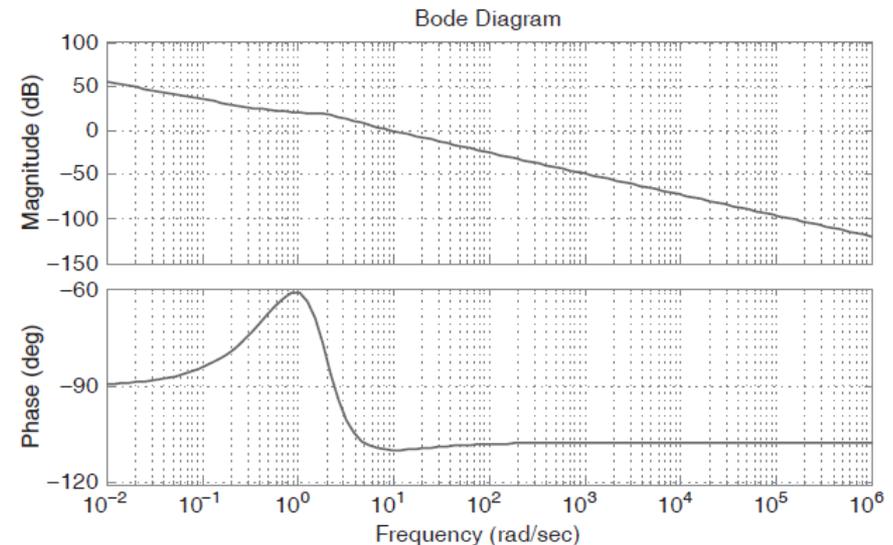


Figure: Bode plot with FO[PD] controller

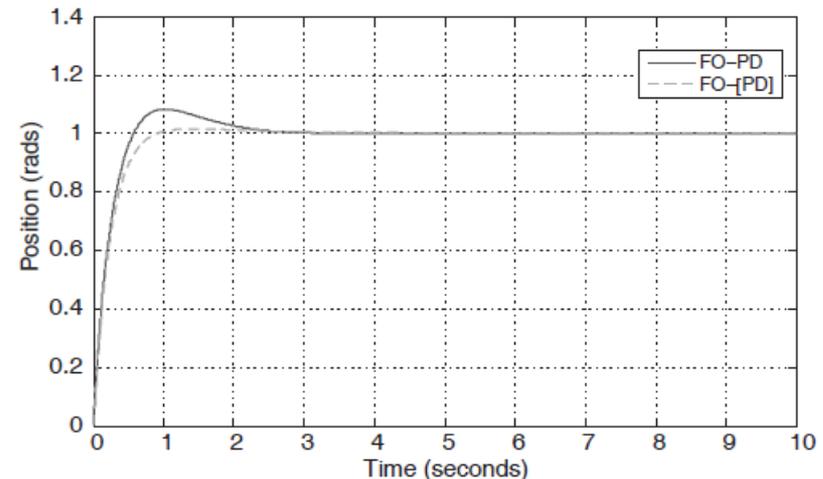


Figure: Step responses comparison with two FO controllers

## Summary and review

- FO PD
- FO [PD] } For IO position system
- FO [PD] Robust to time constant for IO position system with time delay
- FO PD
- FO [PD] } For FO position system

**This is the end of session III**

**Questions?**